

Если выделить двама безконечно близкими поперечными сечениями элемент стержня длиной δx , то силы инерции, соответствующие вращению этого элемента, дадут нам моменты равный

$$- \frac{J \gamma}{g} \frac{\partial^2 y}{\partial x \partial t^2} \delta x \dots \dots \dots (c)$$

При составлении дифференциального ур-я для изогнутой оси стержня нужно принять во внимание не только непрерывно распределенную нагрузку, соответствующую силам инерции поступательного движения, но и непрерывно распределенные моменты, определяемые ϕ -ой (c). В таком случае влияние изгибающего момента вдоль оси стержня представится так

$$\frac{\partial M}{\partial x} = Q - \frac{J \gamma}{g} \frac{\partial^2 y}{\partial x \partial t^2}$$

Здесь Q поперечная сила, получающаяся от сил инерции поступательного движения элементов стержня.

Вставляя найденное значение производной $\frac{\partial M}{\partial x}$ в дифференциальное ур-е

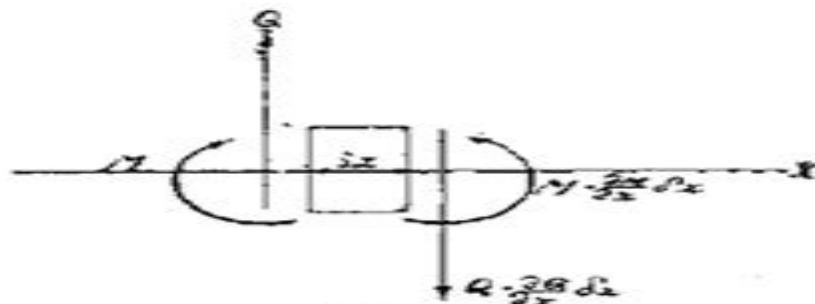
$$EJ \frac{\partial^2 y}{\partial x^2} = - \frac{\partial M}{\partial x},$$

получаем

$$EJ \frac{\partial^4 y}{\partial x^4} = - \frac{\gamma F}{g} \frac{\partial^2 y}{\partial t^2} + \frac{J \gamma}{g} \frac{\partial^4 y}{\partial x^2 \partial t^2} \dots \dots \dots (177)$$

Второй член в правой части ур-я представляет собой поправку, обусловленную поворотом поперечных сечений при колебаниях.

Более точное ур-е мы получим, если примем во внимание не только влияние сил инерции, соответствующих повороту сечений, но также и влияние касательных напряжений на прогиб стержня¹⁾. Выделим из стержня двумя безконечно близкими сечениями элемент длиной δx и составим для него ур-я движения. Положение выделенного элемента определится перемещением y его центра тяжести и углом поворота ϕ относительно нейтральной линии²⁾. Так как прогиб



Черт. 76.

стержня обусловлен не только относительно поворотом поперечных сечений но также и движением их относительно друг друга, то угол наклона кас-

¹⁾ Влияние касательных напряжений на период поперечных колебаний стержня обыкновенно не принимается в расчет, а между тем соответствующая ему поправка по своей величине больше той, которая получается от поперечных сил сечений.

²⁾ При составлении ур-я мы рассматриваем элемент, как абсолютно твердое тело и пренебрегаем искривлением поперечных сечений, поэтому наша поправка лишь приближенно оцелявает влияние касательных сил. Нами совместно с проф. П. Эренфестом получено также точное решение задачи для колеса прямоугольного поперечного сечения. Это решение, как и цитированное нами выше (стр. 184) решение Рессламмгофа для круглого цилиндра, показывают, что приводимое здесь приближенное решение с достаточной точностью учитывает влияние касательных сил.

Elasticity Timoshenko Solution Manual

G Orfield



Elasticity Timoshenko Solution Manual:

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