

Если выделить двумя бесконечно близкими поперечными сечениями элемент стержня длиной δx , то силы инерции, соответствующие вращению этого элемента, дадут нам моментъ равный

$$- \frac{J \gamma}{g} \frac{\partial^2 y}{\partial x \partial t^2} \delta x \dots \dots \dots (c)$$

При составлении дифференціального ур-ія для изогнутой оси стержня нужно принять во вниманіе не только непрерывно распределенную нагрузку, соответствующую силамъ инерции поступательнаго движенія, но и непрерывно распределенные моменты, опредѣляемые ϕ -ой (c). Въ такомъ случаѣ вліяніе изгибающаго момента вдоль оси стержня представится такъ

$$\frac{\partial M}{\partial x} = Q - \frac{J \gamma}{g} \frac{\partial^2 y}{\partial x \partial t^2}$$

Здѣсь Q поперечная сила, получающаяся отъ силъ инерции поступательнаго движенія элементовъ стержня.

Вставляя найденное значеніе производной $\frac{\partial M}{\partial x}$ въ дифференціальное ур-іе

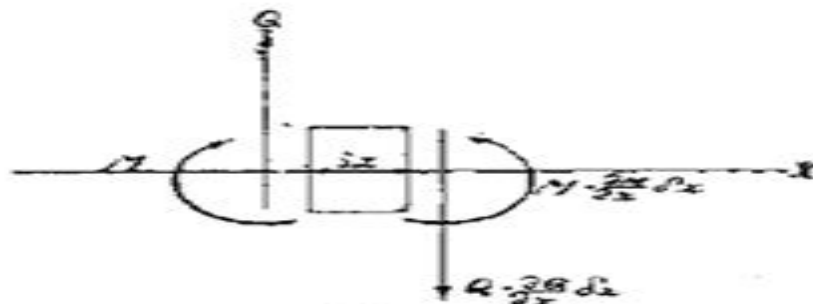
$$EJ \frac{\partial^2 y}{\partial x^2} = - \frac{\partial M}{\partial x},$$

получаемъ

$$EJ \frac{\partial^4 y}{\partial x^4} = - \frac{\gamma F}{g} \frac{\partial^2 y}{\partial t^2} + \frac{J \gamma}{g} \frac{\partial^4 y}{\partial x^2 \partial t^2} \dots \dots \dots (177)$$

Второй членъ въ правой части ур-ія представляетъ собой поправку, обусловленную поворотомъ поперечныхъ сеченій при колебаніяхъ.

Болѣе точное ур-іе мы получимъ, если примемъ во вниманіе не только вліяніе силъ инерции, соответствующихъ повороту сѣченій, но также и вліяніе касательныхъ напряженій на прогибъ стержня¹⁾. Выделимъ изъ стержня двумя бесконечно близкими сѣченіями элементъ длиной δx и составимъ для него ур-іе движенія. Положеніе выделеннаго элемента опредѣлится по размѣщенію y его центра тяжести и угломъ поворота ϕ относительно нейтральной линіи²⁾. Такъ какъ прогибъ



Черт. 76.

стержня обусловленъ не только относительнымъ поворотомъ поперечныхъ сѣченій, но также и движеньемъ ихъ одно относительно другого, то уголъ наклона кас-

¹⁾ Вліяніе касательныхъ напряженій на періодъ поперечныхъ колебаній стержня обыкновенно не принимается въ расчетъ, а между тѣмъ соответствующая ему поправка по своей величинѣ больше той, которая получается отъ поперечнаго сѣченія.

²⁾ При составленіи ур-ія мы рассматриваемъ элементъ, какъ абсолютно твердое тѣло и пренебрегаемъ искривленіемъ поперечныхъ сѣченій, поэтому наша поправка лишь приближенно оцѣниваетъ вліяніе касательныхъ силъ. Нами совместно съ проф. П. Эренфестомъ получено также точное рѣшеніе задачи для колеса прямоугольнаго поперечнаго сѣченія. Это рѣшеніе, какъ и цитированное нами выше (стр. 184) рѣшеніе Рессламмгофа для круглаго цилиндра, показываютъ, что приведенное здѣсь приближенное рѣшеніе съ достаточной точностью учитываетъ вліяніе касательныхъ силъ.

Elasticity Timoshenko Solution Manual

Marcel A. Müller



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